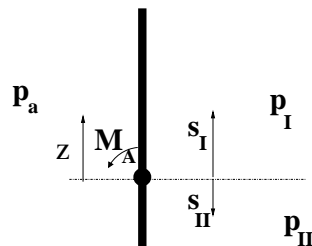


(Name, Matr.-Nr, Unterschrift)

## Klausur Strömungslehre

17. 03. 2007

### 1. Aufgabe



$$\sum M_A = \int_0^{h_1} (p_I - p_a) s_I B ds_I - \int_0^{h_2} (p_{II} - p_a) s_{II} B ds_{II}$$

Hydrostatische Grundgleichung:

$$p(z) + \rho g z = p_a + \rho g (h + h_1)$$

$$\text{mit } s_I = z: p_I(s_I) + \rho g s_I = p_a + \rho g (h + h_1)$$

$$\text{mit } s_{II} = -z: p_{II}(s_{II}) - \rho g s_{II} = p_a + \rho g (h + h_1)$$

$$\begin{aligned} \sum M_A &= \rho g B \left( \left[ (h + h_1) \frac{s_I^2}{2} - \frac{s_I^3}{3} \right]_0^{h_1} - \left[ (h + h_1) \frac{s_{II}^2}{2} + \frac{s_{II}^3}{3} \right]_0^{h_2} \right) \\ &= \rho g B \left[ (h + h_1) \frac{1}{2} (h_1^2 - h_2^2) - \frac{1}{3} (h_1^3 - h_2^3) \right] \end{aligned}$$

$$\text{a) } \sum M_A = 0 \rightarrow h = \frac{2}{3} \frac{h_1^3 + h_2^3}{h_1^2 - h_2^2} - h_1; \quad h = \frac{5}{3}$$

b) Bedingung für das Öffnen der Klappe:

$$d(\sum M_A) > 0$$

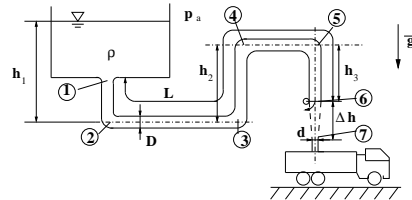
$$d(\sum M_A) = \rho g B \frac{1}{2} (h_1^2 - h_2^2) dh$$

$$\rightarrow h_1^2 - h_2^2 > 0$$

$$\rightarrow dh > 0 \text{ für } d(\sum M_A) > 0.$$

D.h. die Klappe öffnet bei steigendem Wasserspiegel.

## 2. Aufgabe



### a) stationärer Bernoulli: '0'-'6'

$$p_a = p_a + \frac{\rho}{2} v_{stat}^2 + \rho g (-h_1 + h_2 - h_3)$$

$$\rightarrow v_{stat} \sqrt{2g(h_1 - h_2 + h_3)}$$

### instationärer Bernoulli: '0'-'6'

$$p_a = p_a + \frac{\rho}{2} v^2 + \rho g (-h_1 + h_2 - h_3) + \rho \int \frac{\partial v}{\partial t} ds$$

$$\text{mit } \int \frac{\partial v}{\partial t} ds = \frac{dv}{dt} L, \text{ da } D \ll L$$

$$\rightarrow \frac{dv}{dt} = \frac{2g(h_1 - h_2 + h_3) - v^2}{2L} = \frac{v_{stat}^2 - v^2}{2L}$$

$$\rightarrow \int_0^{0.5v_{stat}} \frac{dv}{v_{stat}^2 - v^2} = \int_0^{\Delta T} \frac{dt}{2L}$$

$$\text{mit Hinweis: } \rightarrow \Delta T = \left[ \frac{L}{v_{stat}} \ln \left( \frac{v_{stat} + v}{v_{stat} - v} \right) \right]_0^{0.5v_{stat}}$$

$$\Delta T = \frac{L}{v_{stat}} \ln \left[ \frac{1.5}{0.5} \right] = 1.099 \frac{L}{v_{stat}}$$

### b) aus a): $dt = \frac{2L}{v_{stat}^2 - v^2} dv$

Wasservolumen:

$$V_{Wasser} = \int_0^{\Delta T} v(t) \frac{\pi D^2}{4} dt \rightarrow V_{Wasser} = \int_0^{0.5v_{stat}} \frac{\pi D^2}{4} \frac{2Lv(t)}{v_{stat}^2 - v(t)^2} dv(t)$$

$$\text{mit Hinweis: } V_{Wasser} = -\frac{\pi D^2}{4} \frac{1}{2} 2L \ln[v_{stat}^2 - v(t)^2]_0^{0.5v_{stat}}$$

$$= -\frac{\pi D^2 L}{4} \ln \left[ \frac{v_{stat}^2 - 0}{v_{stat}^2 - 0.25v_{stat}^2} \right] = \frac{\pi D^2 L}{2} \ln \left( \frac{1}{0.75} \right)$$

$$= 0.288 \frac{\pi D^2 L}{4} = 0.072 \pi D^2 L$$

### c) stationärer Bernoulli: '6'-'7'

$$p_a + \rho g \Delta h + \frac{\rho}{2} v_{stat}^2 = p_a + \frac{\rho}{2} v_7^2$$

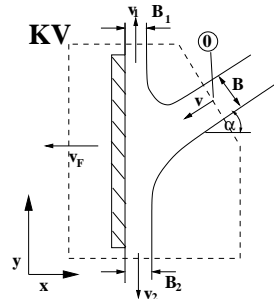
Konti:

$$\frac{\pi D^2}{4} v_{stat} = \frac{\pi d^2}{4} v_7 \rightarrow v_7 = \frac{D^2}{d^2} v_{stat}$$

$$\rightarrow \Delta h = \frac{v_7^2 - v_{stat}^2}{2g} = \frac{v_{stat}^2}{2g} \left( \frac{D^4}{d^4} - 1 \right)$$

$$\Delta h = (h_1 - h_2 + h_3) \left( \frac{D^4}{d^4} - 1 \right)$$

### 3. Aufgabe



- a) Impulssatz in x-Richtung:

$$\frac{dI_x}{dt} = \rho v^2 \cos \alpha B = F_x$$

Die gesuchte Kraft ist  $F = -F_x$ .

- b) verlustfreie Umlenkung, Bernoulli von '0'-'1' und '0'-'2':

$$p_a + \frac{\rho}{2} v^2 = p_a + \frac{\rho}{2} v_1^2$$

$$p_a + \frac{\rho}{2} v^2 = p_a + \frac{\rho}{2} v_2^2$$

$$\rightarrow v_2 = v_1 = v$$

Kontinuität:

$$\rightarrow vB = v_1 B_1 + v_2 B_2 \rightarrow B = B_1 + B_2$$

Impuls in y-Richtung KV 1:

$$\frac{dI_y}{dt} = \rho v^2 \sin \alpha B + \rho v_1^2 B_1 - \rho v_2^2 B_2 = 0$$

$$\rightarrow v^2 \sin \alpha B + v^2 B_1 - v^2 (B - B_1) = 0$$

$$\rightarrow (\sin \alpha - 1)B + 2B_1 = 0$$

$$\rightarrow B_1 = \frac{B}{2}(1 - \sin \alpha)$$

$$\rightarrow B_2 = \frac{B}{2}(1 + \sin \alpha)$$

- c) Impulssatz in x-Richtung für die bewegte Kontrollfläche:

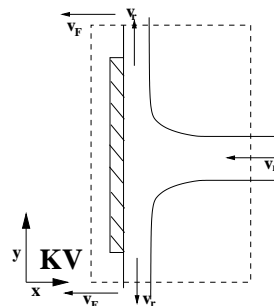
$$\frac{dI_x}{dt} = \int \rho v_a (\vec{v}_r \cdot \vec{n}) dA = F_x$$

$$\rho v_a v_r - \rho v_F v_r \frac{B}{2} - \rho v_F v_r \frac{B}{2} = F_x$$

$$\rightarrow F_x = \rho v(v - v_F)B - \rho v_F(v - v_F)B$$

$$= \rho(v - v_F)^2 B$$

Die gesuchte Kraft  $F = -F_x$ .

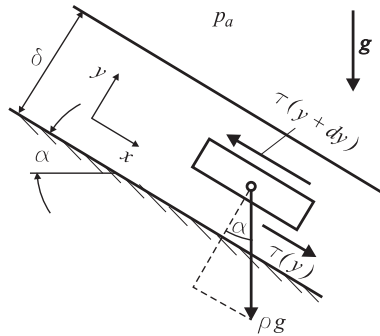


$v_a$  = Absolutgeschw.

$v_r$  = Relativgeschw.

$$v_r = v - v_F$$

#### 4. Aufgabe



a)

Kräftebilanz am Volumenelement in x-Richtung:

$$\left[ \tau - \tau + \frac{d\tau}{dy} dy \right] dx B + \rho g \sin \alpha dx dy B = 0$$

$$\rightarrow \frac{d\tau}{dy} = \rho g \sin \alpha.$$

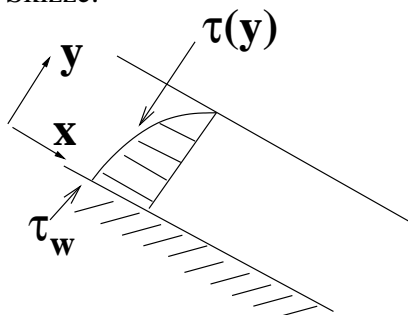
$$\text{Integration } \int d\tau = \int \rho_w \left( 1 - \beta(T_w - T_a) \frac{y}{\delta} \right) g \sin \alpha dy$$

$$\rightarrow \tau = \rho_w g \sin \alpha \left[ y - \beta(T_w - T_a) \frac{y^2}{2\delta} \right] + C_1$$

$$\text{R.B.: } \tau(y = \delta) = 0 \rightarrow C_1 = -\rho_w g \sin \alpha \delta \left[ 1 - \frac{\beta}{2} (T_w - T_a) \right]$$

$$\rightarrow \tau(y) = \rho_w g \sin \alpha \left[ y - \delta - \frac{\beta(T_w - T_a)}{2} \left( \frac{y^2}{\delta} - \delta \right) \right]$$

Skizze:



b)  $\tau(y) = -\eta \frac{du}{dy}$  und mit  $k = \frac{\beta(T_w - T_a)}{2}$

$$\int du = \int \left[ -\frac{\rho_w g \sin \alpha}{\eta} \left( y - \delta - k \left( \frac{y^2}{\delta} - \delta \right) \right) \right] dy$$

$$\rightarrow u(y) = -\frac{\rho_w g \sin \alpha}{\eta} \left( \frac{y^2}{2} - \delta y - k \left( \frac{y^3}{3\delta} - y\delta \right) \right) + C_2$$

$$\text{Randbedingung: } u(y = 0) = 0 \rightarrow C_2 = 0$$

$$\rightarrow u(y) = -\frac{\rho_w g \sin \alpha}{\eta} \left( \frac{y^2}{2} - \delta y - k \left( \frac{y^3}{3\delta} - y\delta \right) \right)$$

c)  $\frac{\dot{m}}{B} = \int_0^\delta \delta(y) u(y) dy$

$$\frac{\dot{m}}{B} = \int_0^\delta -\frac{\rho_w g \sin \alpha}{\eta} \left( 1 - 2k \frac{y}{\delta} \right) \left( \frac{y^2}{2} - \delta y - k \left( \frac{y^3}{3\delta} - y\delta \right) \right) dy$$

## 5. Aufgabe

$$\begin{aligned}
 \text{a) } \bar{\omega} &= \frac{\omega}{\omega_0}, \quad \bar{r} = \frac{r}{R}, \quad \bar{t} = \frac{t}{t_0} \\
 &\rightarrow \frac{\omega_0}{t_0} \frac{\partial \bar{\omega}}{\partial \bar{t}} = \nu R \bar{r} \frac{1}{R} \frac{\partial}{\partial \bar{r}} \left( \frac{1}{R \bar{r}} \frac{\omega_0}{R} \frac{\partial \bar{\omega}}{\partial \bar{r}} \right) \\
 &\rightarrow \frac{\omega_0}{t_0} \frac{\partial \bar{\omega}}{\partial \bar{t}} = \nu \bar{r} \frac{\omega_0}{R^2} \frac{\partial}{\partial \bar{r}} \left( \frac{1}{\bar{r}} \frac{\partial \bar{\omega}}{\partial \bar{r}} \right) \\
 &\rightarrow \frac{\partial \bar{\omega}}{\partial \bar{t}} = \frac{\nu t_0}{R^2} \bar{r} \frac{\partial}{\partial \bar{r}} \left( \frac{1}{\bar{r}} \frac{\partial \bar{\omega}}{\partial \bar{r}} \right) \\
 &\rightarrow K_r = \frac{\nu t_0}{R^2}
 \end{aligned}$$

$$\begin{aligned}
 \text{b) } \omega(R_1, t_1) &= \omega(R, t_0) \\
 &\rightarrow K_r \text{ ist konstant, so dass } K_{r_1} = K_{r_2} \\
 &\rightarrow \frac{\nu t_1}{R_1^2} = \frac{\nu t_0}{R^2} \\
 &\rightarrow \frac{\nu 2t_0}{R_1^2} = \frac{\nu t_0}{R^2} \\
 &\rightarrow R_1 = \sqrt{2} R
 \end{aligned}$$

$$\text{c) } \omega(r, t) = \omega_0 e^{-\frac{r^2}{f(t)}}$$

$$\begin{aligned}
 &\bullet \frac{\partial \omega}{\partial r} = \omega_0 \left( -\frac{2r}{f(t)} \right) e^{-\frac{r^2}{f(t)}} \\
 &\bullet r \frac{\partial}{\partial r} \left( \frac{1}{r} \frac{\partial \omega}{\partial r} \right) = \left( \frac{2r}{f(t)} \right)^2 \omega(r, t) \\
 &\bullet \frac{\partial \omega}{\partial t} = \omega_0 e^{-\frac{r^2}{f(t)}} \left( \frac{r^2}{f^2(t)} \frac{\partial f}{\partial t} \right) = \omega(r, t) \frac{r^2}{f^2(t)} \frac{\partial f}{\partial t}
 \end{aligned}$$

in die DGL eingesetzt:  $\frac{\partial f}{\partial r} = 4\nu$

$$\rightarrow f(t) = 4\nu t + C_1$$

$$\text{Anfangsbedingung: } t = t_0; r = R \rightarrow \omega = \omega_0 e^{-1} = \omega_0 e^{-\frac{R^2}{4\nu t_0 + C_1}}$$

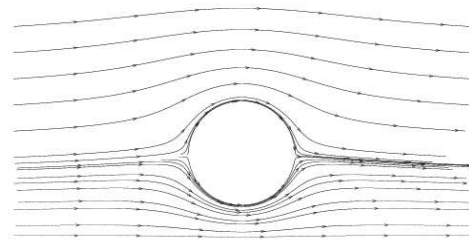
Vergleich der Exponenten ergibt:

$$C_1 = R^2 - 4\nu t_0$$

$$f(t) = 4\nu(t - t_0) + R^2$$

## 6. Aufgabe

a)  $F(z) = u_\infty z + \frac{M}{2\pi} \left[ \frac{1}{z + ib} + \frac{1}{z - ib} \right]$   
 b)



c)  $\Phi(z) = \operatorname{Re}(F(z)) = \operatorname{Re} \left( u_\infty z + \frac{M}{2\pi z} \right) = u_\infty r \cos \theta + \frac{M \cos \theta}{2\pi r}$   
 $v_r = \frac{\partial \Phi}{\partial r} = u_\infty \cos \theta - \frac{M \cos \theta}{2\pi r^2} = u_\infty \cos \theta \left( 1 - \frac{M}{2\pi r^2 u_\infty} \right)$   
 $v_\theta = \frac{1}{r} \frac{\partial \Phi}{\partial \theta} = \frac{1}{r} \left( -u_\infty r \sin \theta - \frac{M \sin \theta}{2\pi r} \right) = -u_\infty \sin \theta \left( 1 + \frac{M}{2\pi r^2 u_\infty} \right)$

Bedingung auf der Kontur:  $v_r(r = R) = 0$

$$\rightarrow u_\infty \cos \theta = \frac{M \cos \theta}{2\pi r^2} \rightarrow M = 2\pi u_\infty R^2$$

$$v_r = u_\infty \cos \theta \left( 1 - \frac{R^2}{r^2} \right), \quad v_\theta = -u_\infty \sin \theta \left( 1 + \frac{R^2}{r^2} \right)$$

d) Druckbeiwert auf der Kontur:

$$c_p = 1 - \frac{v_r^2(r = R) + v_\theta^2(r = R)}{u_\infty^2} \quad \text{mit } v_\theta(r = R) = -u_\infty \sin \theta (1 + 1) = -2u_\infty \sin \theta$$

$$c_p = 1 - \frac{(-2u_\infty \sin \theta)^2}{u_\infty^2} = 1 - 4 \sin^2 \theta$$

## 7. Aufgabe

a) Reibungsfreie Außenströmung:  $\frac{dp}{dx} = -\rho U_a \frac{\partial U_a}{\partial x} \big|_{y=\delta}$  mit  $U_a(x) = Cx^{\frac{1}{3}}$

$$\frac{dp}{dx} = -\rho Cx^{\frac{1}{3}} \frac{1}{3} Cx^{-\frac{2}{3}} = -\frac{1}{3} \rho C^2 x^{-\frac{1}{3}}$$

$$\frac{dp}{dx} < 0 \text{ d.h. beschleunigte Strömung} \rightarrow \text{keine Ablösung}$$

b) Polynomansatz:  $\frac{u(x, y)}{U_a(x)} = a_1 \left(\frac{y}{\delta}\right) + a_2 \left(\frac{y}{\delta}\right)^2 + a_3 \left(\frac{y}{\delta}\right)^3 + a_4 \left(\frac{y}{\delta}\right)^4$

Randbedingungen:

1.)  $\frac{u}{U_a} = 0$  für  $\frac{y}{\delta} = 0$

2.)  $\frac{u}{U_a} = 1$  für  $\frac{y}{\delta} = 1$

3.)  $\frac{\partial^2 \left(\frac{u}{U_a}\right)}{\partial \left(\frac{y}{\delta}\right)^2} = \frac{dp}{dx}$  für  $\frac{y}{\delta} = 0$

4.)  $\frac{\partial \left(\frac{u}{U_a}\right)}{\partial \left(\frac{y}{\delta}\right)} = 0$  für  $\frac{y}{\delta} = 1$

5.)  $\frac{\partial^2 \left(\frac{u}{U_a}\right)}{\partial \left(\frac{y}{\delta}\right)^2} = 0$  für  $\frac{y}{\delta} = 1$

mit 1.) ist identisch erfüllt ...

mit 2.)  $\Rightarrow a_1 + a_2 + a_3 + a_4 = 1$

mit 3.) mit  $U_a(x) = Cx^{\frac{1}{3}}$  und  $\frac{dp}{dx} = -\frac{1}{3} \rho C^2 x^{-\frac{1}{3}}$  (siehe a)) folgt:

$$\rightarrow \frac{\partial^2 \left(\frac{u}{U_a}\right)}{\partial \left(\frac{y}{\delta}\right)^2} \big|_{y/\delta=0} = \eta \frac{U_a}{\delta^2} 2a_2 \text{ folgt } a_2 = -\frac{\rho \delta^2}{6\eta} Cx^{-\frac{2}{3}}$$

mit 4.)  $\Rightarrow a_1 + 2a_2 + 3a_3 + 4a_4 = 0$

mit 5.)  $\Rightarrow 2a_2 + 6a_3 + 12a_4 = 0$

$$\rightarrow 4 = 3a_1 + 2a_2 + a_3$$

$$\rightarrow 0 = 3a_1 + 4a_2 + 3a_3$$

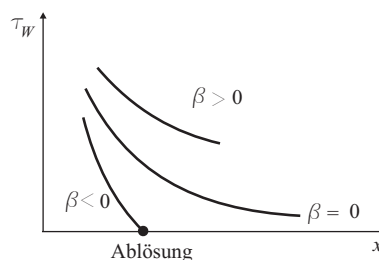
$$\rightarrow 4 = -2a_2 - 2a_3$$

$$\rightarrow a_3 = -2 - 2a_2 = -2 + \frac{\rho \delta^2}{6\eta} Cx^{-\frac{2}{3}}$$

$$\rightarrow a_1 = 2 + \frac{1}{18} \frac{\rho \delta^2}{\eta} Cx^{-\frac{2}{3}}$$

$$\rightarrow a_4 = 1 - \frac{1}{18} \frac{\rho \delta^2}{\eta} Cx^{-\frac{2}{3}}$$

c) Verlauf der Wandschubspannung  $\tau_w$  über  $x$



## 8. Aufgabe

- a) Wenn im engsten Querschnitt Schallgeschwindigkeit erreicht wird, stellt sich der maximale Massendurchsatz  $\dot{m}_{max}$  ein. Also wenn gilt

$$\dot{m}_{max} = \varrho^* u^* A^*$$

$$\varrho^* = \frac{p^*}{RT^*} = \frac{\frac{p_0}{T_0}}{R \frac{T^*}{T_0}} \frac{p_0}{T_0}$$

$$u^* = a^* = \sqrt{\gamma R T^*} = \sqrt{\gamma R \frac{T^*}{T_0} T_0}$$

mit Energiesatz:  $c_p T_0 = c_p T + \frac{u^2}{2}$  und  $c_p = \frac{\gamma R}{\gamma - 1}$

$$\frac{T_0}{T} = 1 + \frac{\gamma - 1}{2} Ma^2$$

kritischer Zustand:  $Ma = Ma^* = 1$

$$\frac{T^*}{T_0} = \left(1 + \frac{\gamma - 1}{2}\right)^{-1} = \frac{2}{\gamma + 1}$$

mit der Isentropenbeziehung folgt

$$\frac{p^*}{p_0} = \left(\frac{T^*}{T_0}\right)^{\frac{\gamma}{\gamma - 1}} = \left(\frac{2}{\gamma + 1}\right)^{\frac{\gamma}{\gamma - 1}}$$

$$\begin{aligned} \dot{m}_{max} &= \left(\frac{2}{\gamma + 1}\right)^{\left(\frac{\gamma}{\gamma - 1} - 1\right)} \frac{p_0}{RT_0} \sqrt{\gamma R \frac{2}{\gamma + 1} T_0} A_D \\ &= A_D \frac{p_0}{\sqrt{RT_0}} \sqrt{\gamma} \left(\frac{2}{\gamma + 1}\right)^{\frac{\gamma + 1}{2(\gamma - 1)}} \end{aligned}$$

b)  $\dot{m}_{max} = \varrho_1 u_1 A_1 \rightarrow A_1 = \frac{\dot{m}_{max}}{\varrho_1 u_1}$

$$\frac{\varrho_1}{\varrho_0} = \left(\frac{T_1}{T_0}\right)^{\frac{1}{\gamma - 1}}; u_1 = Ma_1 \sqrt{\gamma R T_1}$$

$$\varrho_1 = \frac{p_0}{RT_0} \left(1 + \frac{\gamma - 1}{2} Ma_1^2\right)^{-\frac{1}{\gamma - 1}}$$

$$\rightarrow A_1 = A_D \frac{\frac{p_0}{\sqrt{RT_0}} \sqrt{\gamma} \left(\frac{2}{\gamma + 1}\right)^{\frac{\gamma + 1}{2(\gamma - 1)}}}{\frac{p_0}{RT_0} \left(1 + \frac{\gamma - 1}{2} Ma_1^2\right)^{\left(-\frac{1}{\gamma - 1} - \frac{1}{2}\right)} \sqrt{\gamma R T_0} Ma_1}$$

$$\rightarrow A_1 = A_D \frac{1}{Ma} \left[ \left(\frac{2}{\gamma + 1}\right) \left(1 + \frac{\gamma - 1}{2} Ma_1^2\right) \right]^{\frac{\gamma + 1}{2(\gamma - 1)}}$$

c)  $u_1 = \sqrt{\gamma R T_0} Ma_1 \left(1 + \frac{\gamma - 1}{2} Ma_1^2\right)^{-\frac{1}{2}}$