

Potentialtheory

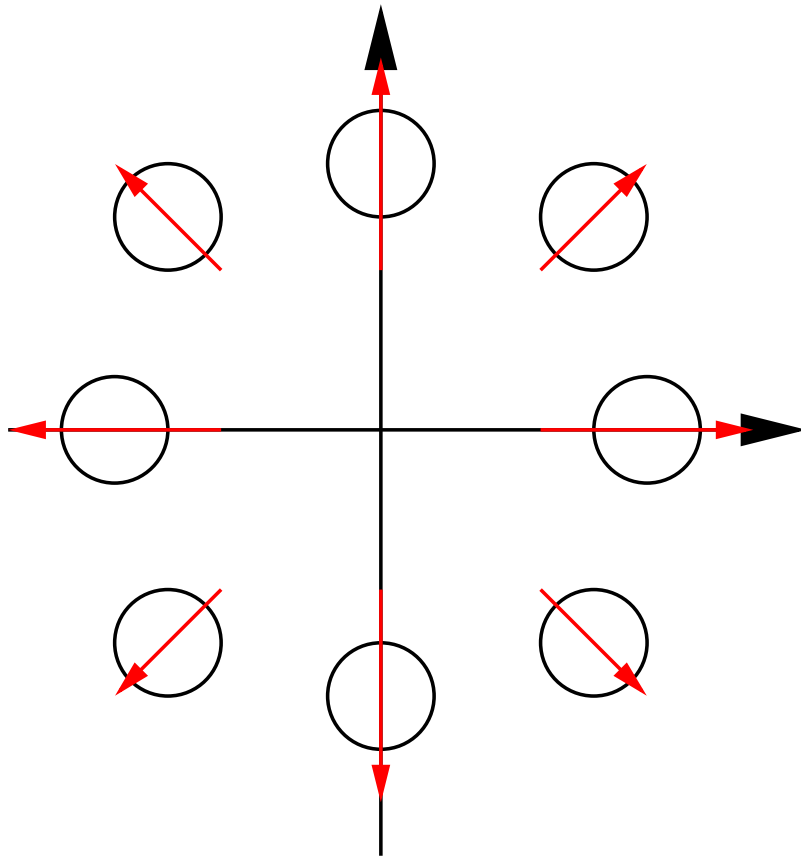
Assumptions: frictionless, rotationless
 2-dimensional (plane)
 incompressible, steady flow

no rotation: $\vec{\omega} = \vec{0}$

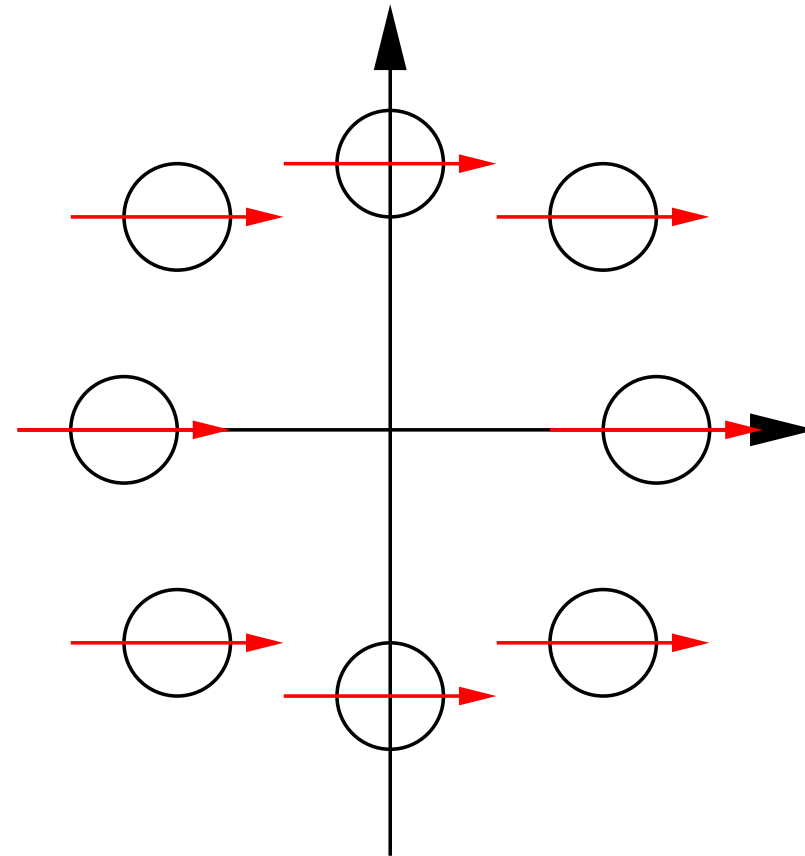
$$\vec{\omega} = \frac{1}{2} \text{rot } \vec{v} = \frac{1}{2} \nabla \times \vec{v} = \frac{1}{2} \begin{pmatrix} w_y - v_z \\ u_z - w_x \\ v_x - u_y \end{pmatrix}$$

2-dimensional flow $\omega_x = \omega_y = 0$

$$\rightarrow \omega_z = \frac{1}{2}(v_x - u_y) = \frac{1}{2} \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) = 0$$



rotational flow



rotation free

Problem 4.1: $\text{rot}(\text{grad } f) = \vec{0}$

Potential theory

if $\omega_z = 0 \rightarrow$ a function Φ exists with the attribute

$$\vec{v} = \nabla \underbrace{\Phi}_{\text{Potential}} \rightarrow \begin{pmatrix} u \\ v \end{pmatrix} = \begin{pmatrix} \frac{\partial \Phi}{\partial x} \\ \frac{\partial \Phi}{\partial y} \end{pmatrix}$$

\rightarrow continuity (2-d, steady, incompressible)

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = \nabla \cdot \vec{v} \rightarrow \boxed{\nabla^2 \Phi = \Delta \Phi = 0}$$

$$\rightarrow \boxed{\frac{\partial^2 \Phi}{\partial x^2} + \frac{\partial^2 \Phi}{\partial y^2} = 0}$$

Potential theory

lineare differential equation

→ the principle of Superposition can be used

if Φ_1, Φ_2 are solutions of the equation, then $C_1 \cdot \Phi_1, C_2 \cdot \Phi_2$ und $C_1 \cdot \Phi_1 + C_2 \cdot \Phi_2$ are also solution of the equation

Stream function : $u = \frac{\partial \Psi}{\partial y}; v = -\frac{\partial \Psi}{\partial x}$ fullfills the continuity equation

$$\omega = 0 \quad \boxed{\nabla^2 \Psi = \Delta \Psi = 0}$$

$\Phi_x = \Psi_y; \Phi_y = -\Psi_x$ → lines of constant Φ and Ψ are perpendicular to each other

Potential theory

$\Phi = \text{const} \rightarrow$ Isopotential lines

$\Psi = \text{const} \rightarrow$ Streamlines

Φ and Ψ are used to describe flowfields around solid bodies

The contour is a special stream line

\rightarrow velocity vector is parallel to the wall

But: The no-slip condition cannot be fulfilled
(frictionless, no roatation)

\rightarrow Drag forces and shear stresses cannot be
berechnet computed

Potential theory

- complex numbers

$$z = x + iy = r e^{i\varphi} = r(\cos \varphi + i \sin \varphi)$$

$$\begin{array}{l} x = r \cos \varphi \\ y = r \sin \varphi \end{array} \quad \Leftrightarrow \quad \begin{array}{l} r = \sqrt{x^2 + y^2} \\ \varphi = \tan^{-1} \left(\frac{y}{x} \right) \end{array}$$

- complex velocity

$$w = u + iv$$

- conjugate complex velocity

$$\bar{w} = u - iv$$

Potential theory

complex Potential function

complex stream function

$$F(z) = \int \bar{w} dz = \Phi(x, y) + i \Psi(x, y)$$

→ Laplace equation

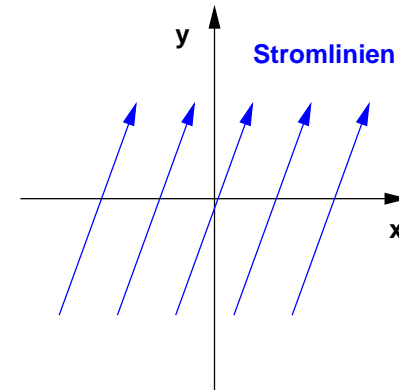
$$\left(\frac{\partial^2 \Phi}{\partial x^2} + \frac{\partial^2 \Phi}{\partial y^2} \right) + i \left(\frac{\partial^2 \Psi}{\partial x^2} + \frac{\partial^2 \Psi}{\partial y^2} \right) = 0$$

$$\bar{w} = u - iv = \frac{dF}{dz}$$

Parallel flow: $F(z) = (u_\infty - i v_\infty) z$

$$\Phi = u_\infty x + v_\infty y$$

$$u = u_\infty$$



$$\Psi = u_\infty y - v_\infty x$$

$$v = v_\infty$$

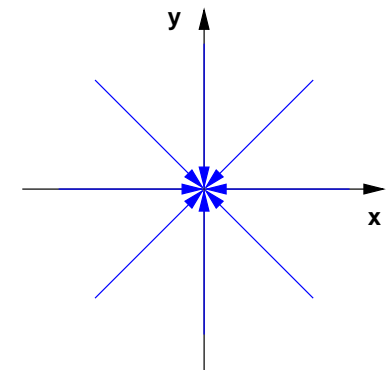
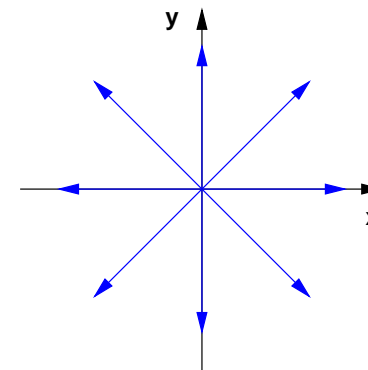
Source, Sink: $F(z) = \frac{E}{2\pi} \ln z$

$$\Phi = \frac{E}{2\pi} \ln r$$

$$u = \frac{E}{2\pi} \frac{x}{x^2 + y^2}$$

$$\Psi = \frac{E}{2\pi} \varphi$$

$$v = \frac{E}{2\pi} \frac{y}{x^2 + y^2}$$



Singularities

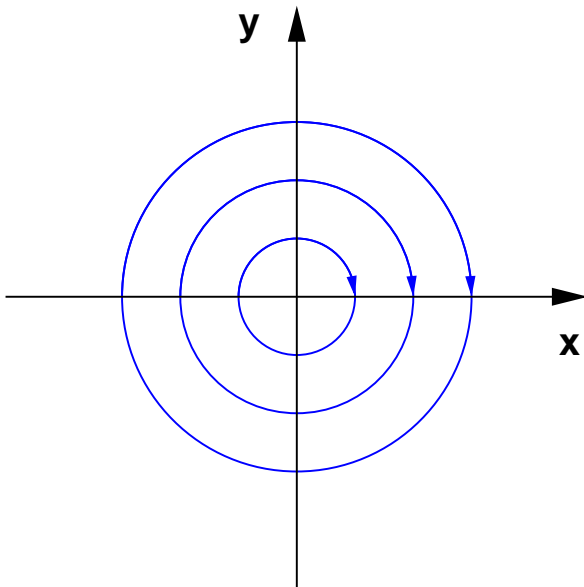
Potential vortex: $F(z) = \frac{\Gamma}{2\pi} i \ln z$

$$\Phi = -\frac{\Gamma}{2\pi} \tan^{-1} \left(\frac{y}{x} \right)$$

$$u = \frac{\Gamma}{2\pi} \frac{y}{x^2 + y^2}$$

$$\Psi = \frac{\Gamma}{2\pi} \ln \sqrt{x^2 + y^2}$$

$$v = -\frac{\Gamma}{2\pi} \frac{x}{x^2 + y^2}$$



Singularities

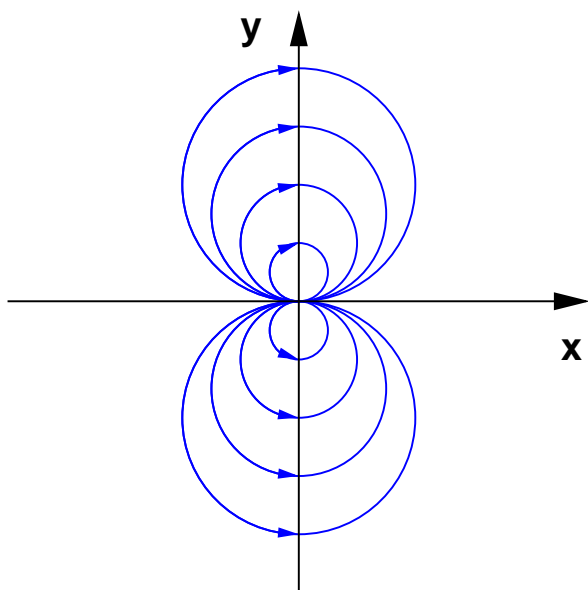
Dipole: $F(z) = \frac{m}{z}$

$$\Phi = \frac{mx}{x^2 + y^2}$$

$$\Psi = -\frac{my}{x^2 + y^2}$$

$$u = m \frac{y^2 - x^2}{(x^2 + y^2)^2}$$

$$v = -m \frac{2xy}{(x^2 + y^2)^2}$$



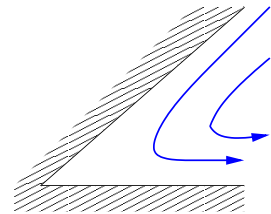
Singularities

Corner flow: $F(z) = \frac{a}{n} z^n \quad (n \in \mathbf{R}, a \in \mathbf{C})$

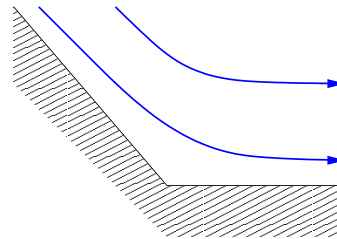
$$\Phi = \frac{a}{n} r^n \cos n\varphi$$

$$\Psi = \frac{a}{n} r^n \sin n\varphi$$

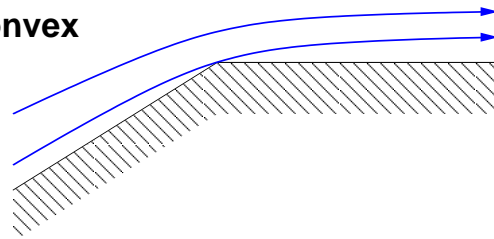
spitzer Winkel



konkav



konvex



Singularities

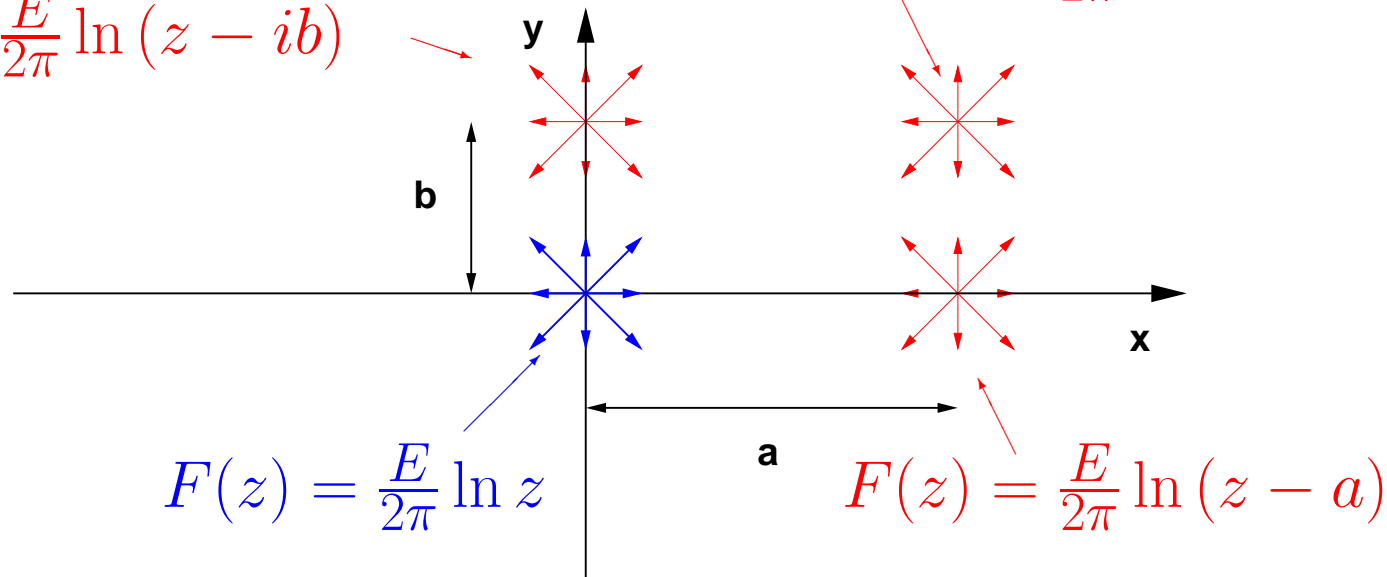
The center of the singularities in the origin of the coordinate system

→ Offset

Example

$$F(z) = \frac{E}{2\pi} \ln(z - ib)$$

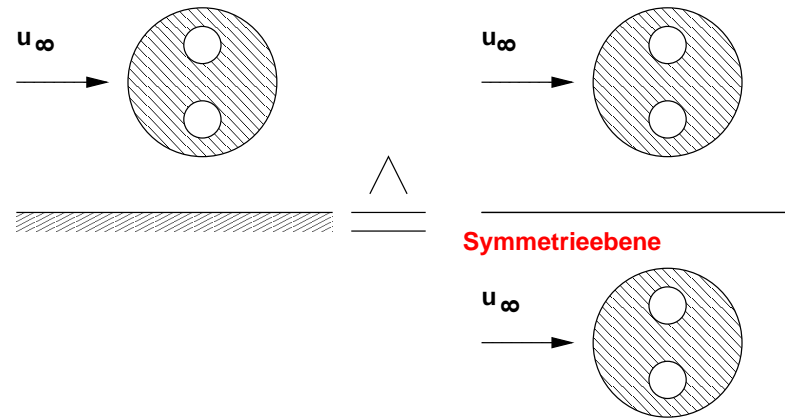
$$F(z) = \frac{E}{2\pi} \ln(z - a - ib)$$



Potential theory

Simulation of walls

by mirroring



- Usually, contours are represented by stagnation point streamlines
 - localisation of stagnation points ($u = v = 0$)
 - Computation of Ψ in the stagnation point
 - sketch of the streamlines

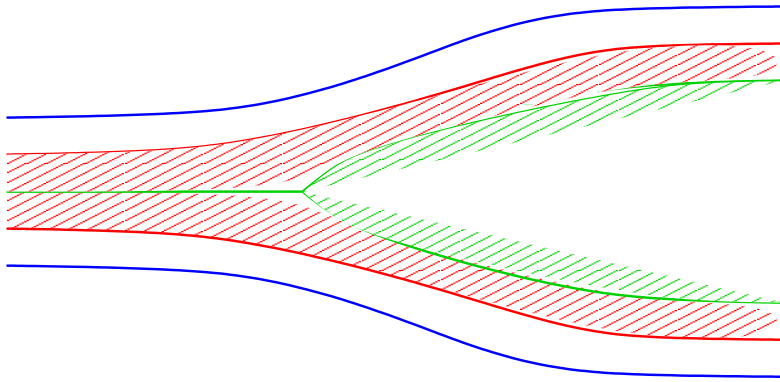
$$\Psi_k(x, y) = \Psi_k(x_s, y_s) = \text{const.}$$

$$\rightarrow y_s = f(x_s)$$

$$\rightarrow r_s = f'(y_s)$$

Potentialtheorie

- streamlines do not intersect
 → any streamline can represent a contour
 usually $u_w \neq 0$



- Bernoulli equation is valid

$$p_0 = p_\infty + \frac{1}{2}\rho(u_\infty^2 + v_\infty^2) = p + \frac{1}{2}\rho(u^2 + v^2) = \text{const.}$$

– computation of $c_p = \frac{p - p_{\text{ref}}}{\frac{\rho}{2}u_{\text{ref}}^2} = \frac{\frac{1}{2}\rho u_{\text{ref}}^2 - \frac{1}{2}\rho \vec{v}^2}{\frac{\rho}{2}u_{\text{ref}}^2} = 1 - \frac{\vec{v}^2}{u_{\text{ref}}^2}$

A planar flow is described by the stream function $\psi = \left(\frac{U}{L}\right)xy$. The pressure in $x_{ref} = 0, y_{ref} = 1 \text{ m}$ is $p_{ref} = 10^5 \text{ N/m}^2$.

$$U = 2 \text{ m/s} \quad L = 1 \text{ m} \quad \rho = 10^3 \text{ kg/m}^3$$

a) Proof, if the flow has a potential!

Determine

b) the stagnation points, the pressure coefficient, and the lines of constant total velocity

c) the velocity and the pressure at $x_1 = 2\text{m}, y_1 = 2\text{m}$,

d) the coordinates of a particle at $t = 0.5\text{s}$, if it passes at $t = -0$ the point x_1, y_1 ,

e) the pressure difference between these two points.

f) Sketch the stream lines.

a) Given: stream function $\Psi = \frac{U}{L}xy$

Φ exists, if $\vec{\omega} = \vec{0}$

planar flow \rightarrow 2-dimensional $\rightarrow \omega_x = \omega_y = 0$

$$\rightarrow \omega_z = \frac{1}{2}(v_x - u_y) = \frac{1}{2} \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) = 0$$

$$\left. \begin{array}{l} u = \frac{\partial \Psi}{\partial y} = \frac{U}{L}x \rightarrow \frac{\partial u}{\partial y} = 0 \\ v = -\frac{\partial \Psi}{\partial x} = -\frac{U}{L}y \rightarrow \frac{\partial v}{\partial x} = 0 \end{array} \right\} \omega_z = 0$$

→ The flow is frictionless and the potential exist
 Φ exists → computation of Φ

$$1.) \quad u = \frac{\partial \Phi}{\partial x} \rightarrow \Phi = \int u \, dx + f_1(y) + C_1$$

$$2.) \quad v = \frac{\partial \Phi}{\partial y} \rightarrow \Phi = \int v \, dy + f_2(x) + C_2$$

$$1.) \quad \Phi(x, y) = \int \frac{U}{L} x \, dx + f_1(y) + C_1$$

$$2.) \quad \Phi(x, y) = \int -\frac{U}{L} y \, dy + f_2(x) + C_2$$

$$1.) \quad \Phi(x, y) = \frac{U x^2}{L 2} + f_1(y) + C_1$$

$$2.) \quad \Phi(x, y) = -\frac{U y^2}{L 2} + f_2(x) + C_2$$

Comparison between 1.) and 2.)

$$\underbrace{\frac{U x^2}{L 2}} + \underbrace{f_1(y)} + C_1 = -\underbrace{\frac{U y^2}{L 2}} + \underbrace{f_2(x)} + C_2$$

$$f_1(y) = -\frac{U y^2}{L 2}; \quad f_2(x) = \frac{U x^2}{L 2}; \quad C_1 = C_2 = C$$

$$\rightarrow \Phi = \frac{U}{2L}(x^2 - y^2) + C$$

complex potential $F(z)$

$$F(z) = F(x + iy) = \Phi(x, y) + i\Psi(x, y)$$

$$= \frac{U}{2L}(x^2 - y^2) + i\frac{U}{L}xy$$

$$= \frac{U}{2L}(x^2 + 2ixy - y^2)$$

$$= \frac{U}{2L}z^2$$

Sketch of the flow field

- Stagnation points \rightarrow stagnation streamlines
- asymptotic streamlines
 $x, y \rightarrow \infty$; $x, y \rightarrow 0$
- direction of the flow

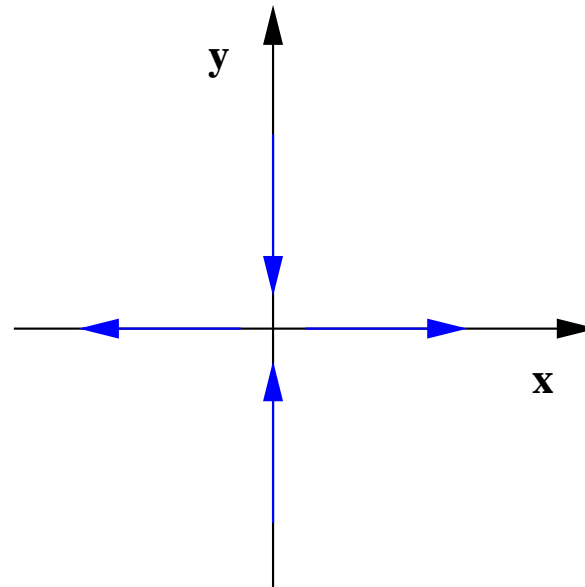
Stagnation points: $\vec{v} = \vec{0} : u = v = 0$

$$u = \frac{U}{L}x, \quad v = -\frac{U}{L}y \rightarrow (x_s, y_s) = (0, 0)$$

14.5

Additionally: $u = 0$ on the y -axis

$v = 0$ on the x -axis



$\Psi = \text{const.}$

$$\Psi = \frac{U}{L}xy = \text{const.}$$

$$\rightarrow y = \frac{L}{U}\text{const.}\frac{1}{x} = \frac{C}{x} \text{ for } x \neq 0$$

$$x = \frac{L}{U}\text{const.}\frac{1}{y} = \frac{C}{y} \text{ for } y \neq 0$$

\rightarrow Hyperbola

Stream lines: $\Psi = \text{const.}$

$$\Psi = \frac{U}{L}xy = \text{const.}$$

$$\rightarrow y = \frac{L}{U}\text{const.}\frac{1}{x} = \frac{C}{x} \text{ for } x \neq 0$$

$$x = \frac{L}{U}\text{const.}\frac{1}{y} = \frac{C}{y} \text{ for } y \neq 0$$

→ Hyperbola

Stagnation stream line

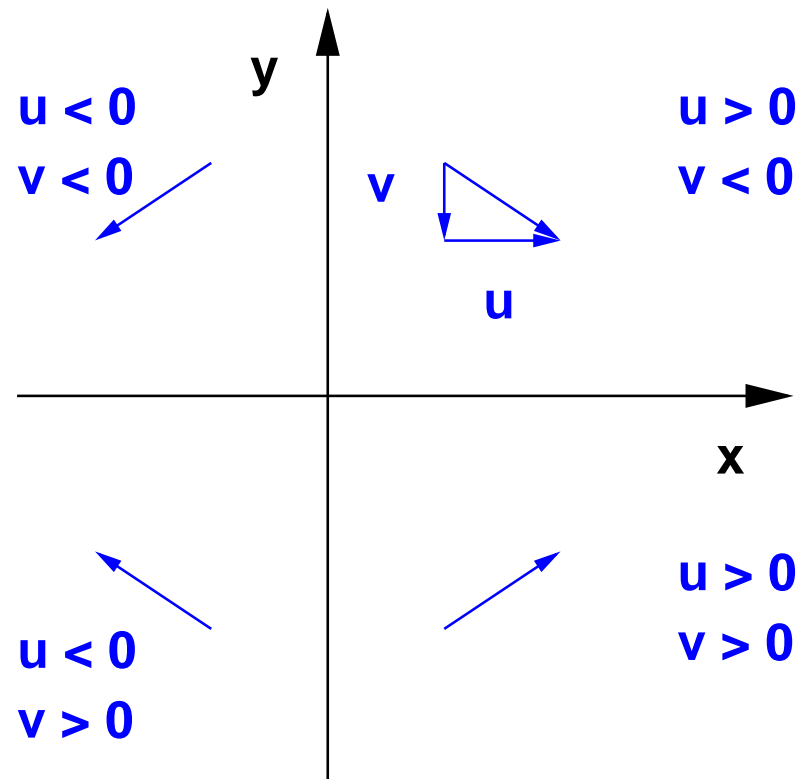
$$\Psi_{sp} = \frac{U}{L}x_{sp}y_{sp} = 0 \quad | \quad \text{problem dependant}$$

$$\Psi = 0 \rightarrow x = 0 \text{ or } y = 0$$

14.5

→ x -Axis and y -Axis are stagnation streamlines

Direction $u = \frac{U}{L}x$, $v = -\frac{U}{L}y$



Pressure coefficient

$$c_p = \frac{p - p_{\text{ref}}}{\frac{\rho}{2} v_{\text{ref}}^2} = 1 - \frac{\vec{v}^2}{v_{\text{ref}}^2} = 1 - \frac{u^2 + v^2}{u_{\text{ref}}^2 + v_{\text{ref}}^2}$$

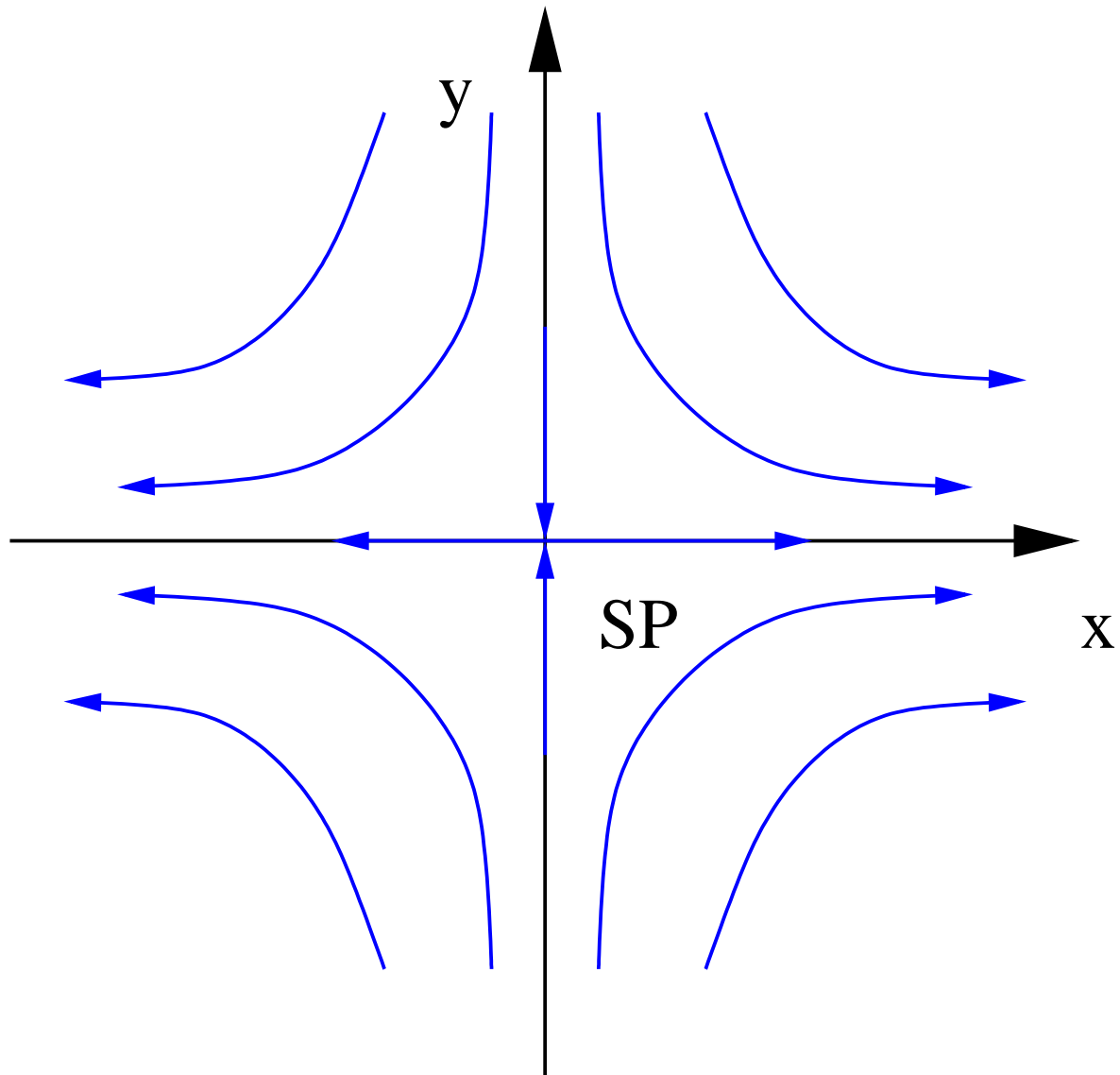
$$\left. \begin{array}{l} u = \frac{U}{L}x \\ v = -\frac{U}{L}y \end{array} \right\} c_p = 1 - \frac{x^2 + y^2}{x_{\text{ref}}^2 + y_{\text{ref}}^2}$$

Lines of constant velocity

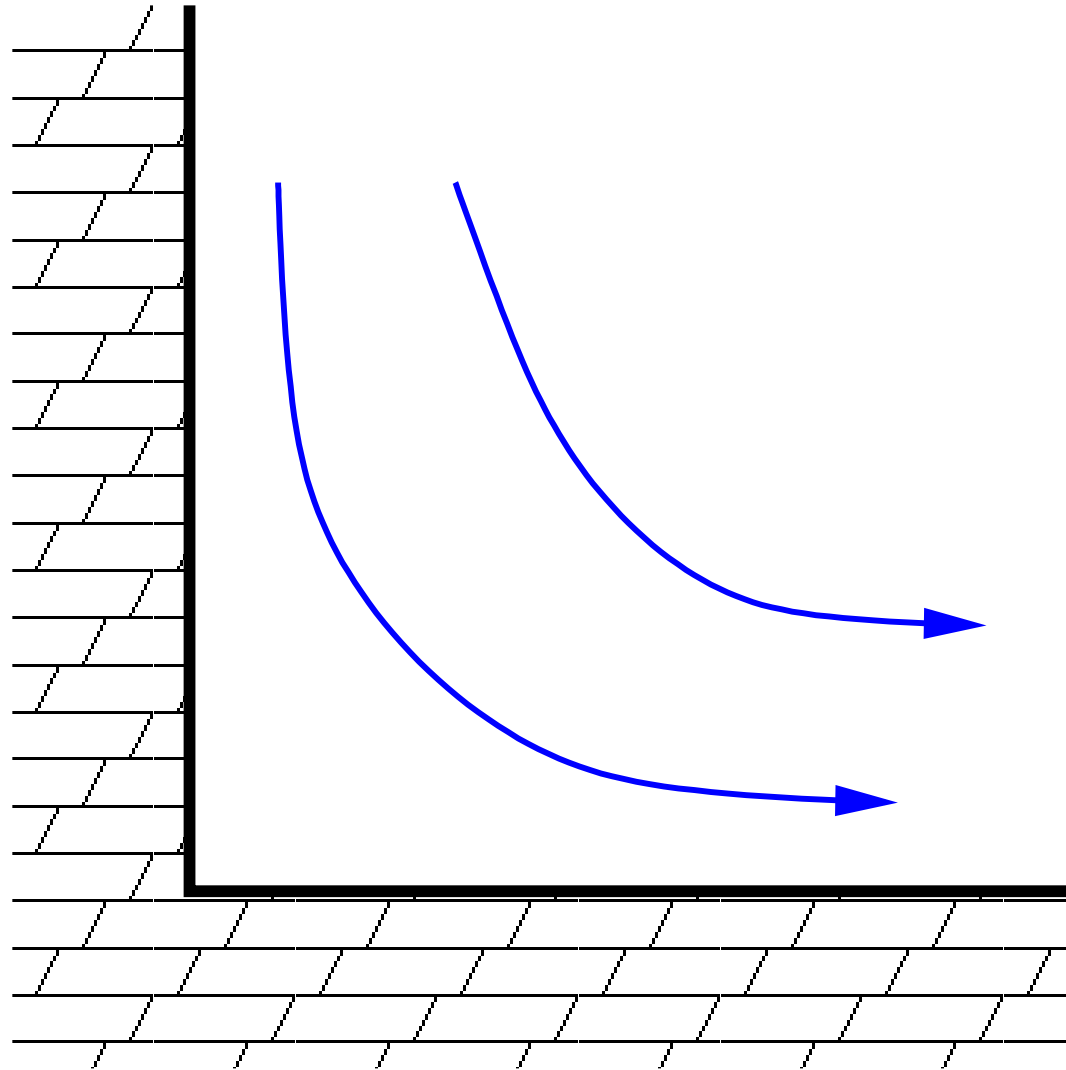
$$|\vec{v}| = \sqrt{u^2 + v^2} = \text{const.}$$

$$= \sqrt{\left(\frac{U}{L}x\right)^2 + \left(-\frac{U}{L}y\right)^2} \rightarrow x^2 + y^2 = \left(\frac{L\vec{v}}{U}\right)^2$$

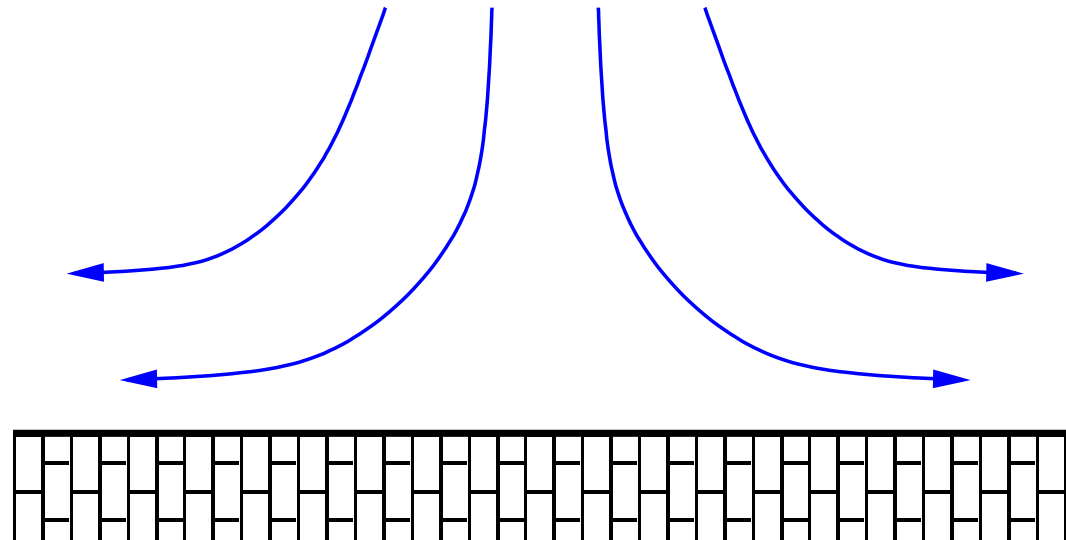
Circles with the radius $R = \frac{L\vec{v}}{U}$



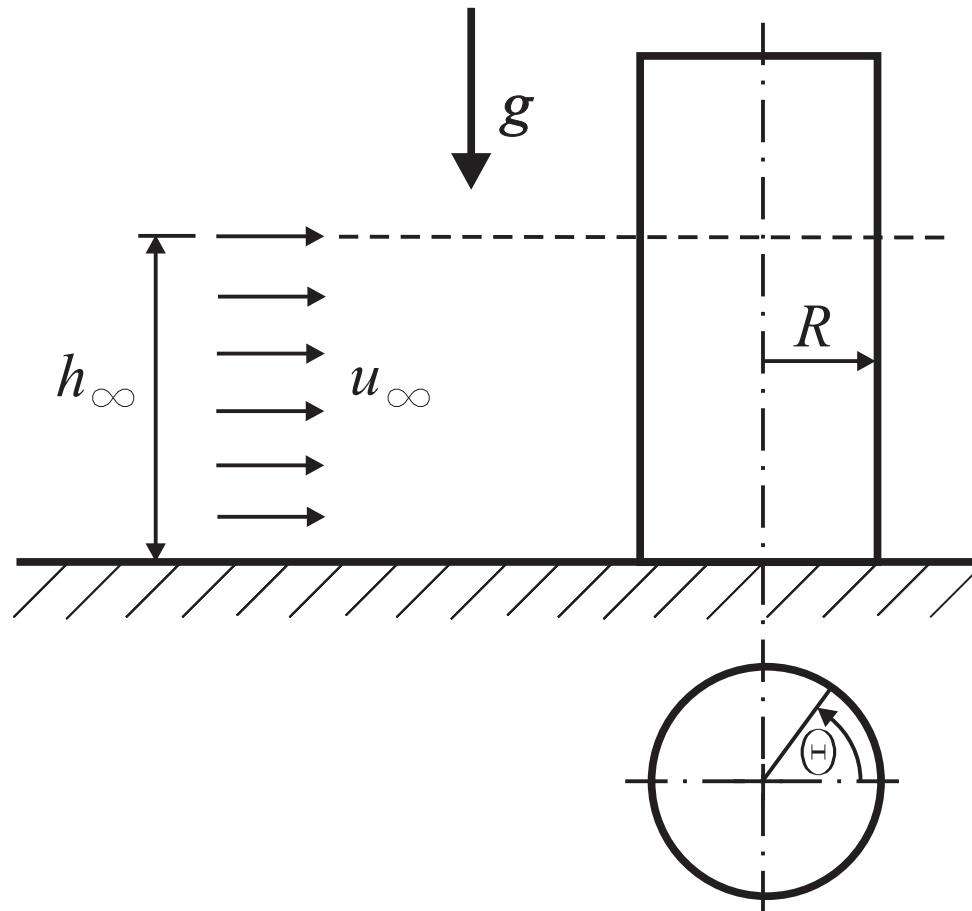
90° corner flow



plane stagnation point flow



A bridge pylon with a circular cross-section is flown against with the velocity u_∞ . Far away from the pylon the water depth is h_∞ .



$$u_{\infty} = 1 \text{ m/s} \quad h_{\infty} = 6 \text{ m} \quad R = 2$$

$$m \quad \rho = 10^3 \text{ kg/m}^3 \quad g = 10 \text{ m/s}^2$$

Determine

- the water depth at the pylon wall as a function of θ ,
- the water depth in the stagnation points,
- the smallest water depth over the ground.

a) Circular cylinder: Dipole + Parallel flow

$$F(z) = u_{\infty}z + \frac{M}{2\pi z} = u_{\infty}z + \frac{R^2 u_{\infty}}{z}$$

$$\rho g h_{\infty} + \frac{\rho}{2} u_{\infty}^2 = \rho g h(\theta) + \frac{\rho}{2} \vec{v}^2$$

$$v_{\theta} = \frac{1}{r} \frac{\partial \Phi}{\partial \theta} = - \left(u_{\infty} + \frac{R^2 u_{\infty}}{r^2} \right) \sin \theta$$

$$r = R : \vec{v}^2 = v_{\theta}^2 = 4u_{\infty}^2 \sin^2 \theta$$

$$h(\theta) - h_{\infty} = \frac{u_{\infty}^2}{2g} (1 - 4 \sin^2 \theta)$$

b) Stagnation points: $\theta = 0$ and $\theta = \pi$

$$h = h_{\infty} + \frac{u_{\infty}^2}{2g} = 6.05 \text{ m}$$

c)

$$\theta_{min} = \frac{\pi}{2}, \frac{3\pi}{2}$$

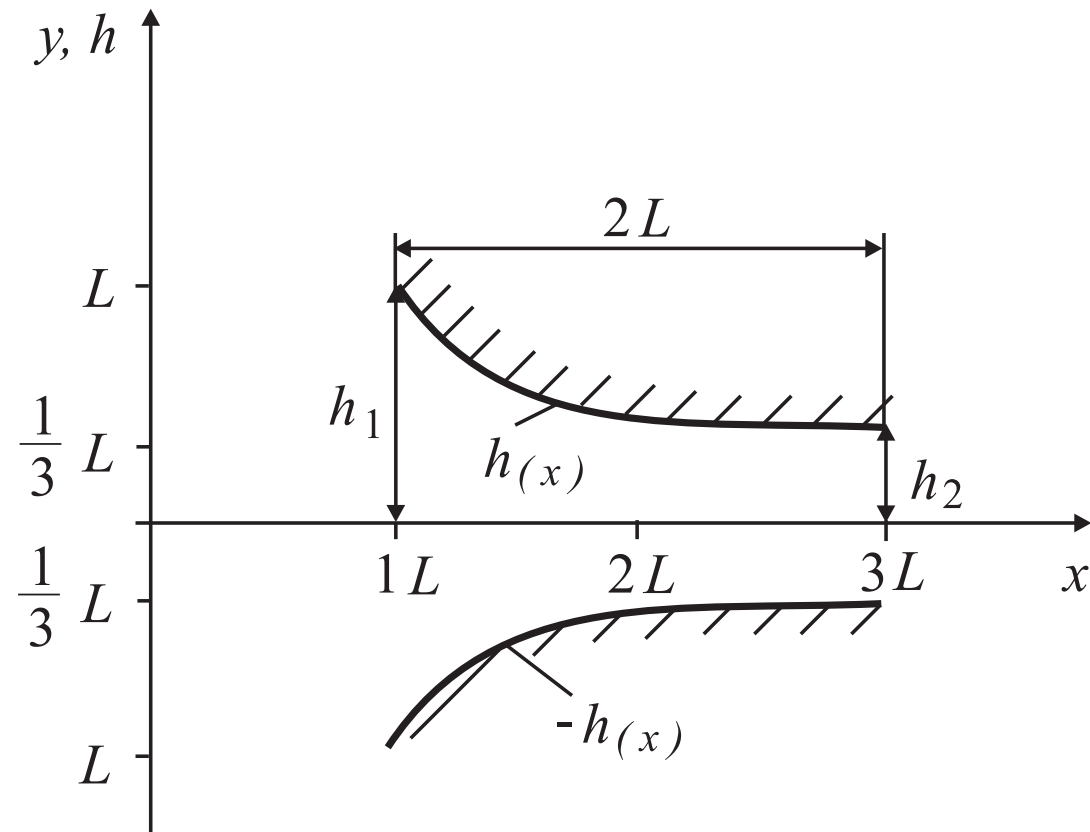
$$h_{min} = h_{\infty} - \frac{3u_{\infty}^2}{2g} = 5.85 \text{ m}$$

The stream function $\psi(x, y)$ for the flow of an incompressible fluid through the sketched plane nozzle is given.

$$\psi(x, y) = \frac{y}{h(x)} u_{\infty} L$$

Given: u_{∞} , L , B , $h_1 = L$, $h_2 = \frac{1}{3}L$

- Determine the upper and the lower contour $h(x)$ such that the flow can be described with the potential theory,
- Compute the velocity distribution $u(x, y)$ and $v(x, y)$.
- Determine the volume flux for a nozzle with the width B .



a)

Condition: $\omega = 0 \rightarrow \frac{\partial^2 \psi}{\partial x^2} + \frac{\partial^2 \psi}{\partial y^2} = 0$

$$\frac{\partial^2 \psi(x, y)}{\partial y^2} = 0 \rightarrow \frac{\partial^2 \psi}{\partial x^2} = \frac{\partial}{\partial x} \left(y u_{\infty} L \frac{\partial}{\partial x} \left(\frac{1}{h(x)} \right) \right) = 0$$

$$\frac{\partial}{\partial x} \left(-y u_{\infty} L \frac{h'(x)}{h^2(x)} \right) = 0 \rightarrow \frac{d}{dx} \left(\frac{h'(x)}{h^2(x)} \right) = 0$$

After 2 integrations: $-\frac{1}{h(x)} = C_1 x + C_2$

BC:

$$x = L, \quad h = L \quad \Longrightarrow \quad -\frac{1}{L} = C_1 L + C_2$$

$$x = 3L, \quad h = \frac{1}{3}L \quad \Longrightarrow \quad -\frac{3}{L} = 3C_1 L + C_2$$

$$\Longrightarrow \quad C_2 = 0, \quad C_1 = -\frac{1}{L^2} \quad \Longrightarrow \quad h(x) = \frac{L^2}{x}$$

b)

$$u = \frac{\partial \psi}{\partial y}; \quad v = -\frac{\partial \psi}{\partial x}; \quad \psi = \frac{u_\infty}{L} xy$$

$$u = u_\infty \frac{x}{L}; \quad v = -u_\infty \frac{y}{L}$$

c)

$$\dot{V} = \psi_{(y=h)} - \psi_{(y=-h)}|_{x=L}$$

$$\dot{V} = (h_1 + h_1)Bu_\infty$$

$$\dot{V} = 2U_\infty LB$$

14.3

The stream function is given $\psi = \psi_1 + \psi_2$

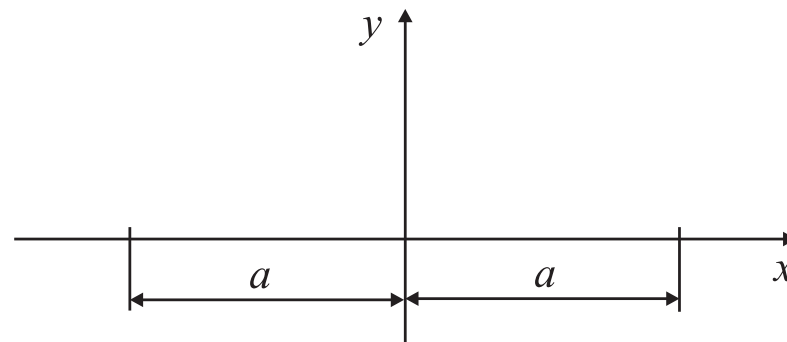
with $\psi_1 = -\frac{\Gamma}{2\pi} \ln \sqrt{(x-a)^2 + y^2}$

$$\psi_2 = -\frac{2\Gamma}{2\pi} \ln \sqrt{(x+a)^2 + y^2}$$

Given: $a, \Gamma > 0$

Determine

- the coordinates of the stagnation point,
- the pressure coefficient on the x -axis $c_p(x, y = 0)$ such, that $c_p = 0$ in the origin of the coordinate system.



a) Condition for the stagnation point: $u = 0, v = 0$

$$u = \frac{\partial \psi}{\partial y} = -\frac{\Gamma}{2\pi} \left(\frac{y}{(x-a)^2 + y^2} + \frac{2y}{(x+a)^2 + y^2} \right)$$

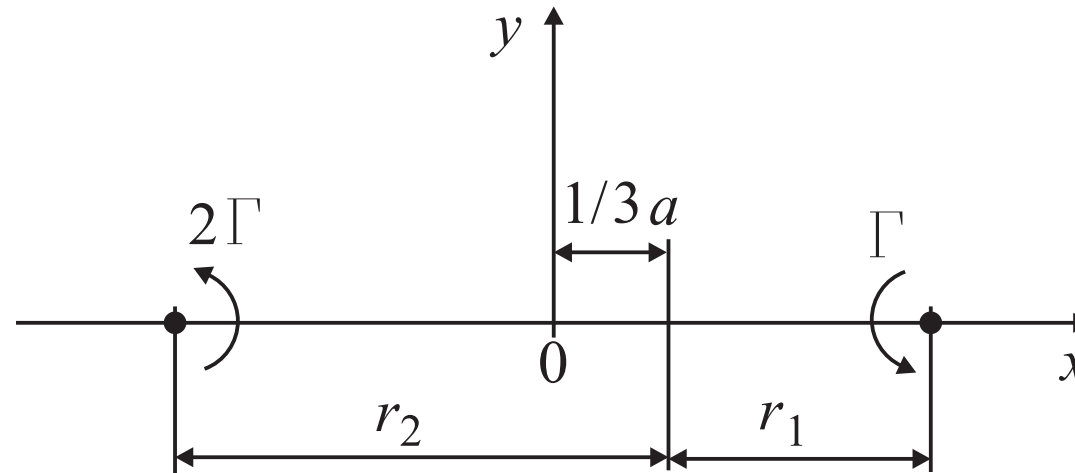
$$u = -\frac{\Gamma}{2\pi} y \left(\frac{1}{(x-a)^2 + y^2} + \frac{2}{(x+a)^2 + y^2} \right) = 0 \quad \text{if } y = 0$$

$$v = -\frac{\partial \psi}{\partial x} = \frac{\Gamma}{2\pi} \left(\frac{x-a}{(x-a)^2 + y^2} + \frac{2(x+a)}{(x+a)^2 + y^2} \right)$$

for $y = 0$: $v = \frac{\Gamma}{2\pi} \left(\frac{1}{x-a} + \frac{2}{x+a} \right) = \frac{\Gamma}{2\pi} \frac{3x-a}{x^2 - a^2} \quad \star)$

$v = 0$, if $3x - a = 0 \rightarrow$ stagnation point: $x_s = \frac{a}{3}, \quad y_s = 0$

alternatively:



$$v_{\theta} = \frac{\Gamma}{2\pi r} : v_{\theta_1} = v_{\theta_2} \rightarrow \frac{\Gamma}{2\pi r_1} = \frac{2\Gamma}{2\pi r_2} \rightarrow r_2 = 2r_1$$

$$2a = r_1 + r_2 \rightarrow 2a = 3r_1 \rightarrow r_1 = \frac{2}{3}a \rightarrow x_s = \frac{a}{3}$$

b)

$$c_p(x, y = 0) = 1 - \frac{u^2 + v^2}{u_{(0,0)}^2 + v_{(0,0)}^2} \quad \text{with } u = 0 \text{ for } y = 0.$$

$$\implies c_p(x, y = 0) = 1 - \frac{v^2}{v_{(0,0)}^2}, \quad v_{(0,0)} = \frac{\Gamma}{2\pi a} \quad (\text{see } \star) \quad \text{i.e. } c_{p(0,0)} = 0$$

$$\text{with } \star) \quad c_p(x, y = 0) = 1 - \left(\frac{\frac{3x - a}{x^2 - a^2}}{\frac{1}{a}} \right)^2$$

$$\implies c_p(x, y = 0) = 1 - \left(\frac{3xa - a^2}{x^2 - a^2} \right)^2 \quad \text{for } x \neq a, -a$$